



SUBTASK 3.1E
EXPERIMENTAL AND NUMERICAL STUDIES OF HIGH INTENSITY
COMBUSTION

Suppression of Knock Intensity by Metal Wire Installation in the
End-gas Region for a Rapid Compression Machine

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Background

To suppress the knocking combustion:

- Promotion of normal flame propagation by
Compactness of the combustion chamber, Central ignition,
Induction of swirl and squish flows, Hydrogen addition, et al.
- Suppression of auto-ignition in end-gas region by
Decrease in the intake air temperature with the inter cooler,
Decrease in the wall temperature with the effective cooling
technique, Reduction of the residual burned gas with the
optimized valve timing, Decrease in the end gas temperature
using the induction of the squish flow or piston speed control, et
al.



Objective

Our new concept for suppression of knocking combustion is metal wire installation in end-gas region.

Expectation:

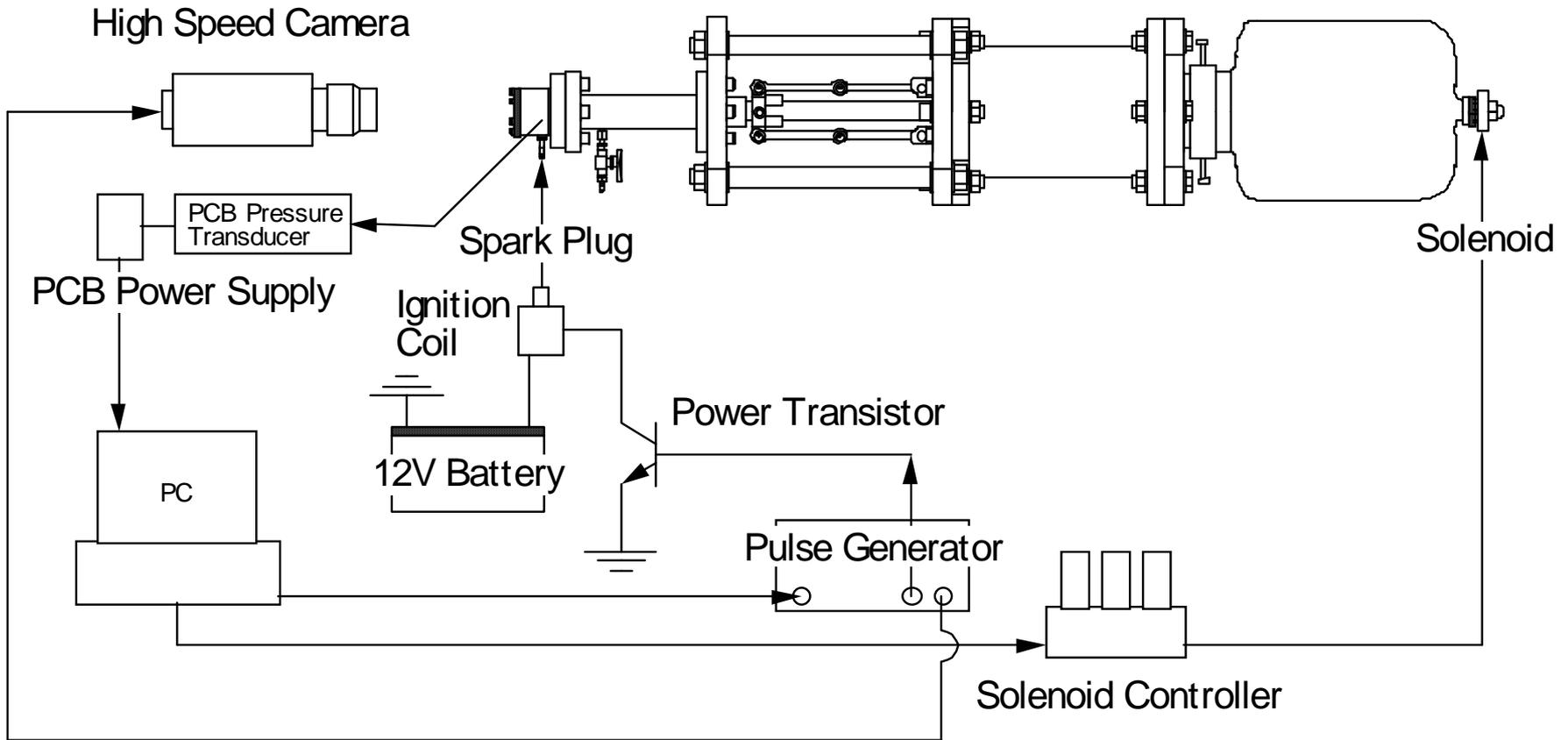
- Suppression of auto-ignition
heat sink, consumption of radical species
- Suppression of pressure vibration (knock intensity)
obstacle
- Others
catalytic effect, et al.

The effect of wire installation on knocking combustion is verified experimentally using a rapid compression machine.



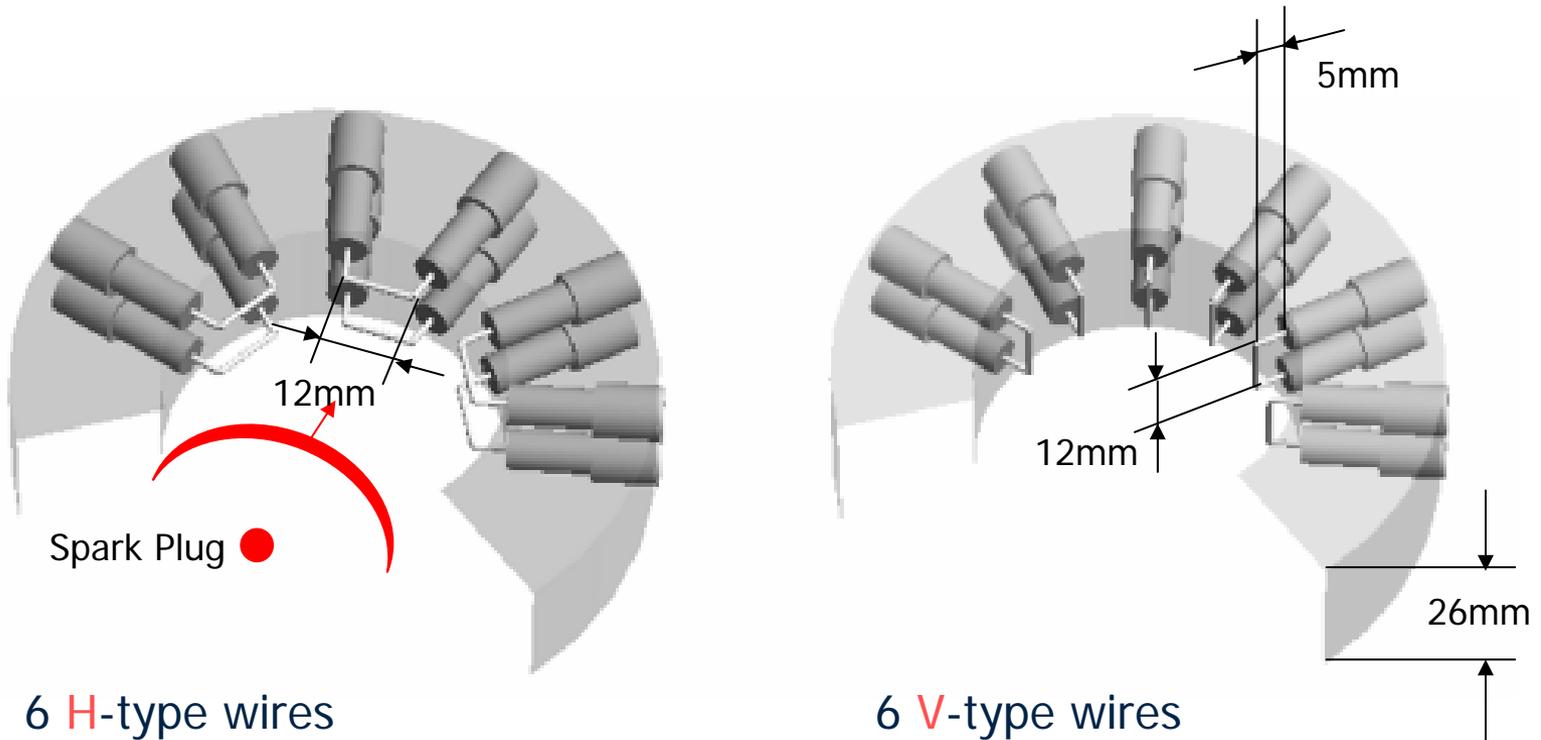
Experimental Setup

Rapid Compression Machine





Combustion Chamber (Type-A)

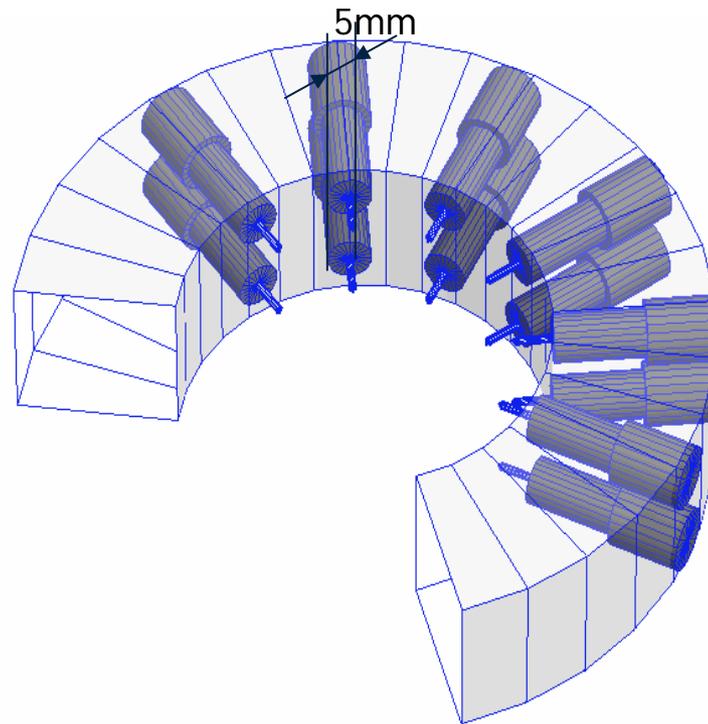


Inner diameter of combustion chamber: 51 mm
Height of combustion chamber at TDC: 26 mm
Wire diameter: 0.9 mm (Ni-Cr)



Configuration of Wire Installation for Combustion (Type-A)

Compression Ratio : 7.5



12N-type wires

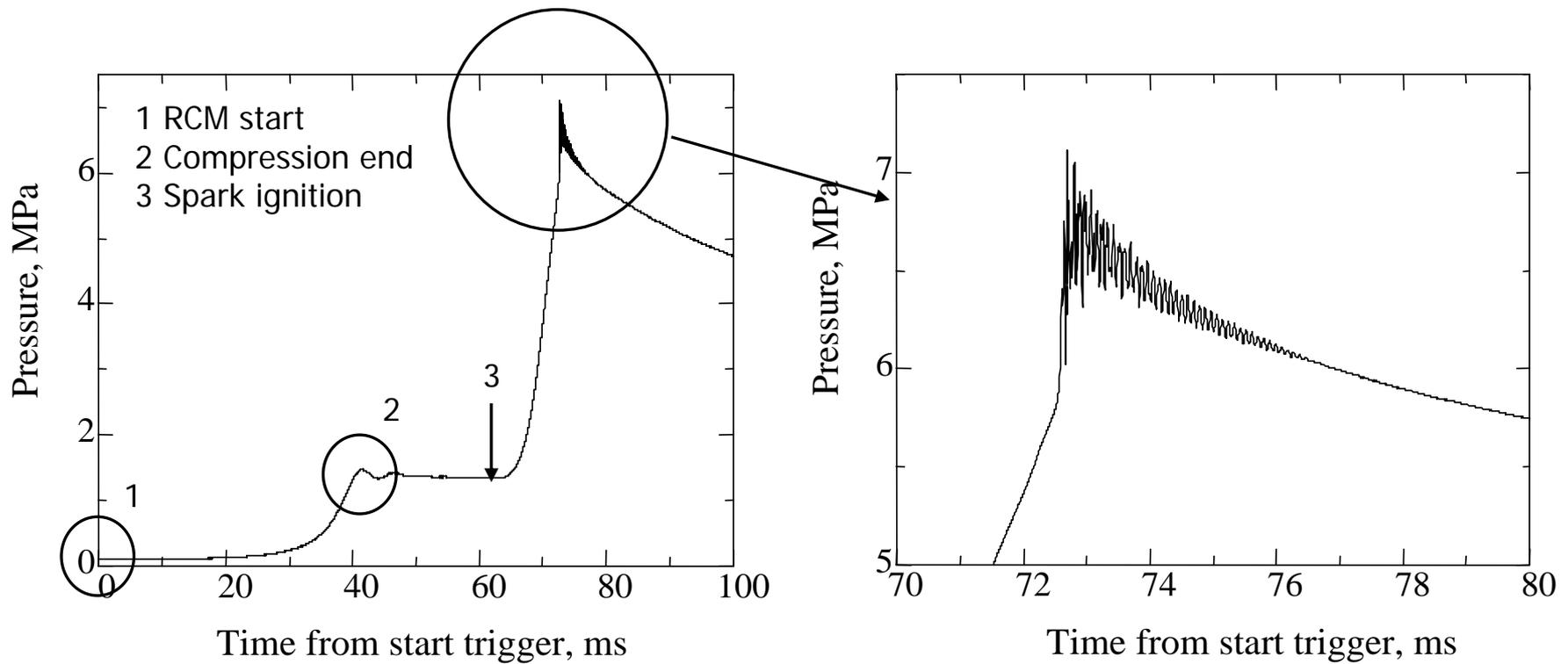


Experimental Condition

Mixture	n-butane - air
Equivalence ratio	1.0
Initial temperature	295K
Initial pressure	0.1 MPa
Compression ratio	5.0, 7.5, 8.5, 9.0

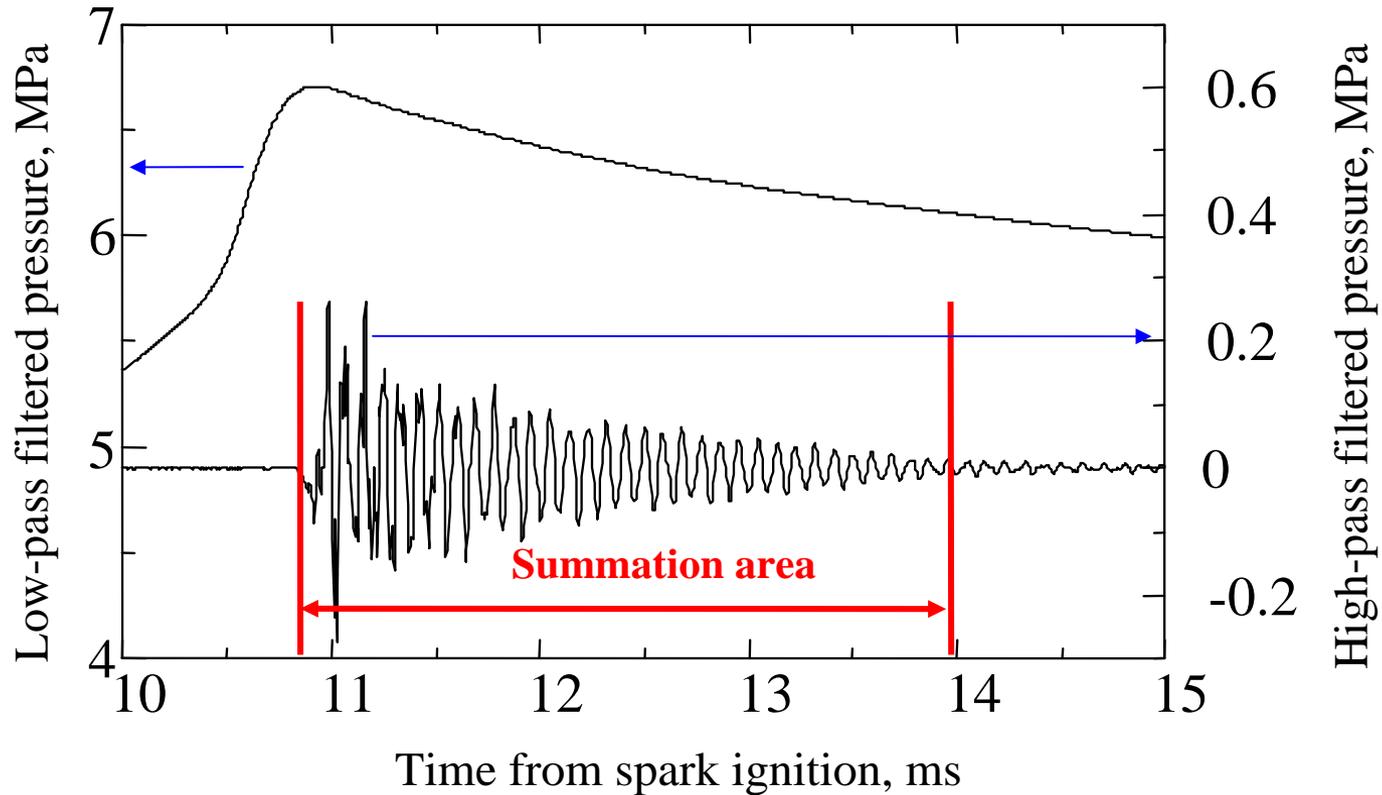


Typical Pressure Trace with Knocking Combustion





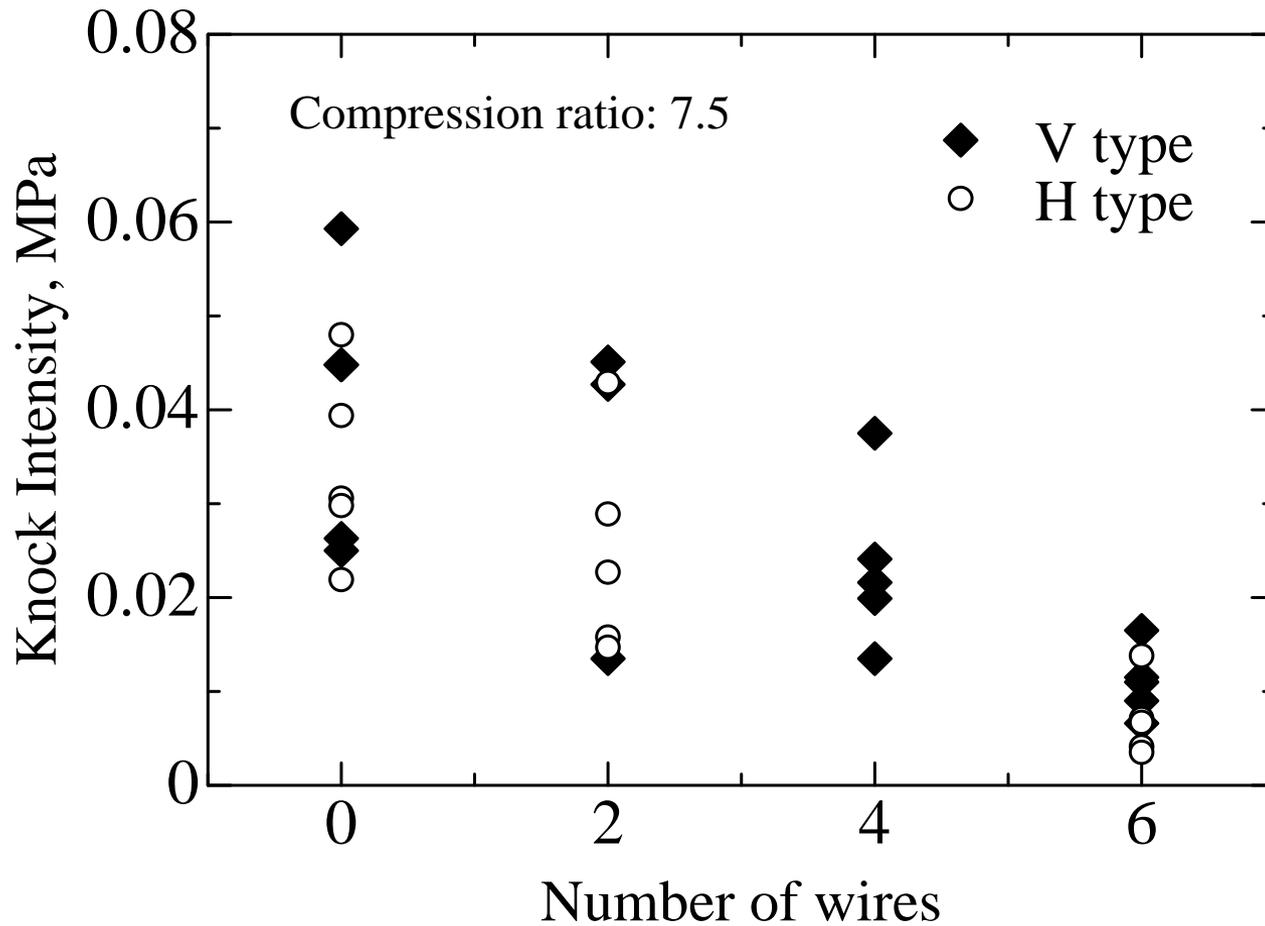
Definition of Knock Intensity



Knock intensity
$$KI = \frac{1}{N} \sum \text{abs}\{p(i)\}$$

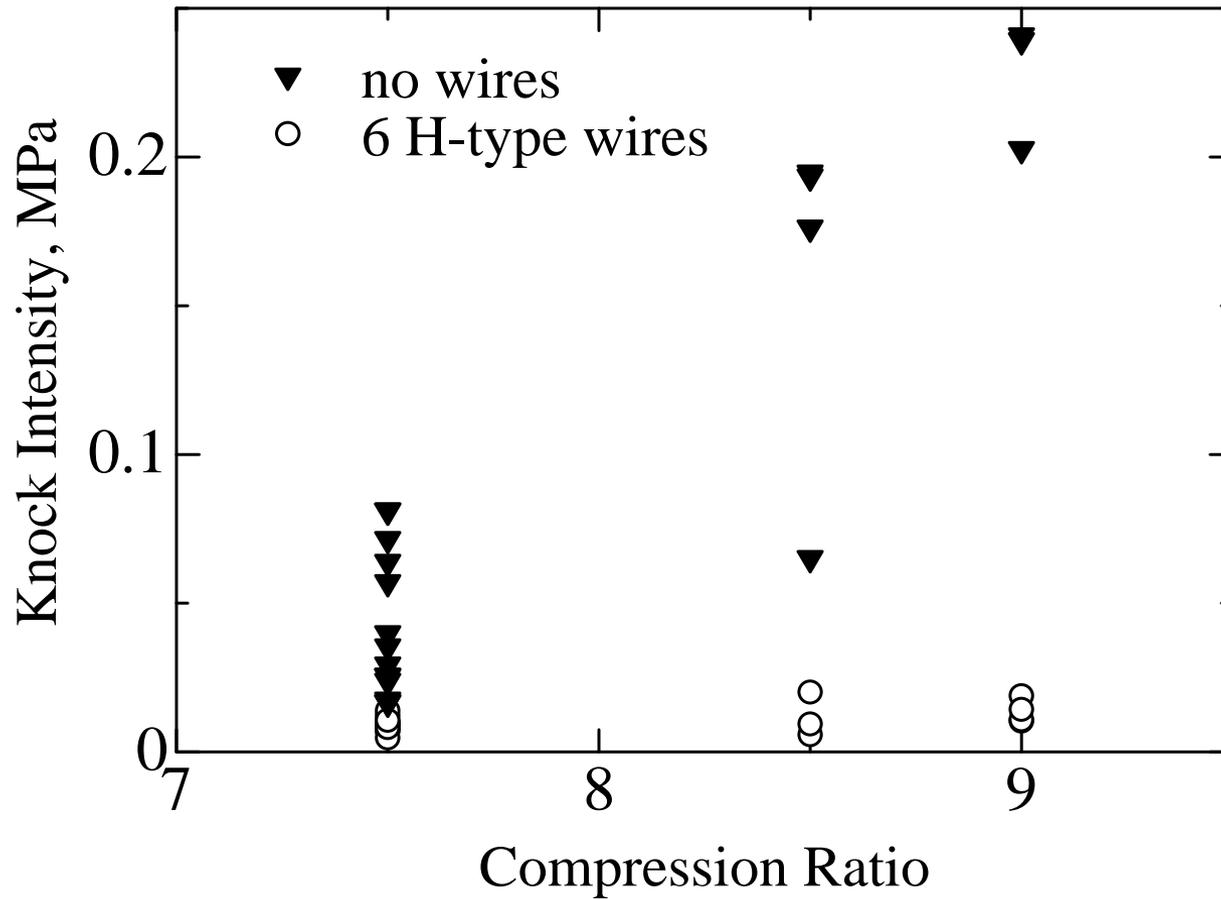


Effect of Configuration and Number of Wires on Knock Intensity





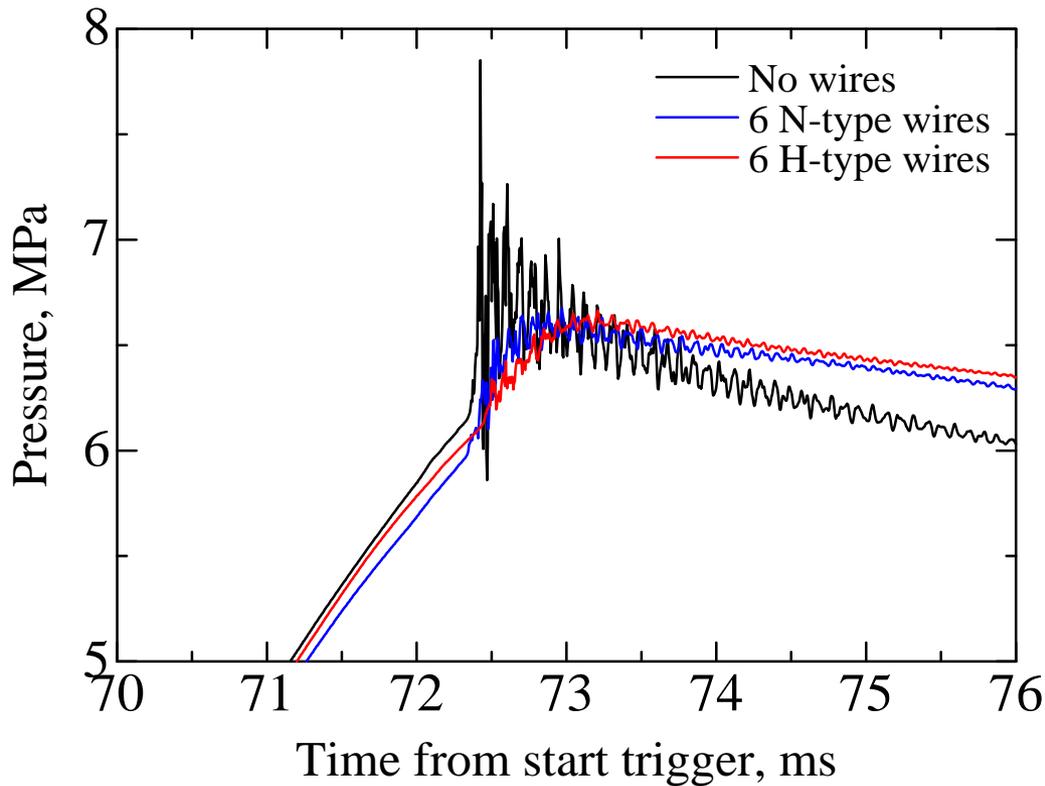
Effect of 6-H Type Wire Installation on Knock Intensity





Effect of N-type Installation on Knock Intensity

Compression Ratio : 7.5



Averaged Knock Intensity

No wires

KI = 0.0438MPa

6 N-type wires

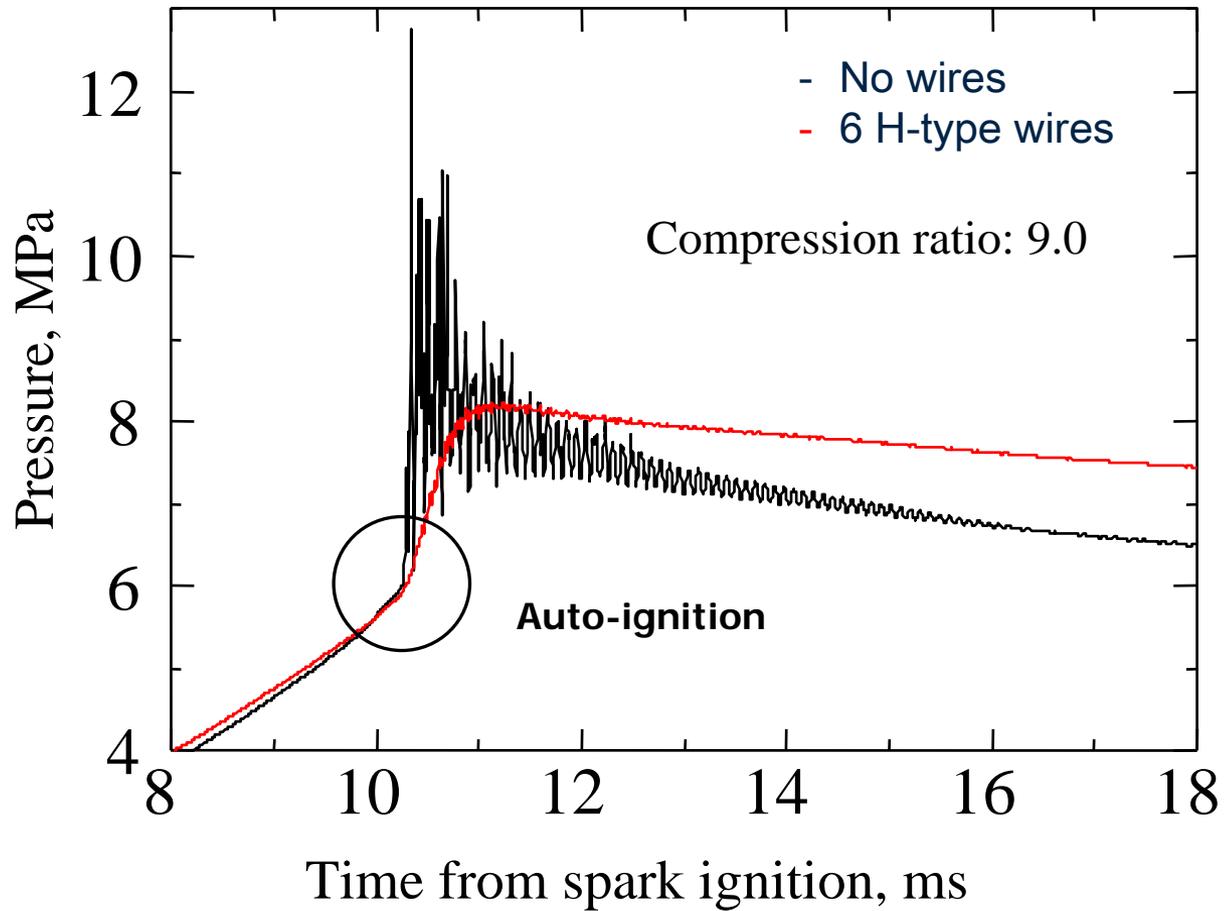
KI = 0.0139MPa

6H-type wires

KI = 0.0078MPa

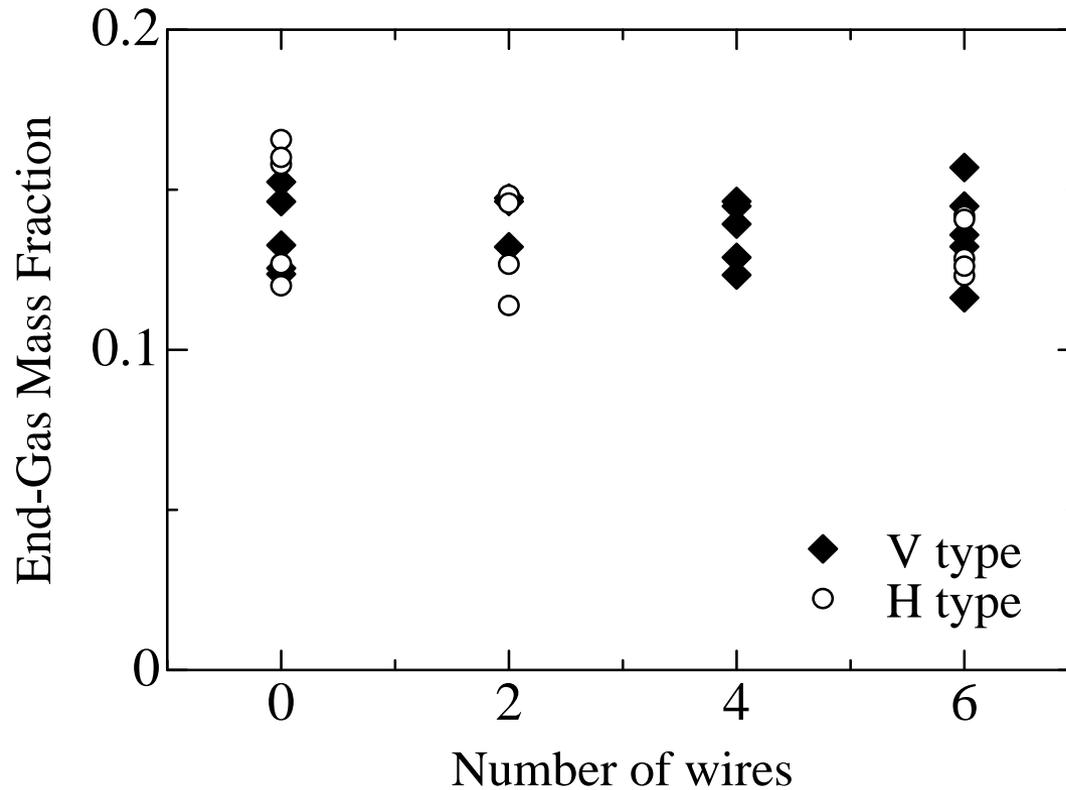


Pressure Traces in the Vicinity of Onset Time of Knocking Combustion





Effect of Wire Installation on End-gas Mass Fraction



$$EMF = \frac{P_e - P_k}{P_e - P_{ig}}$$

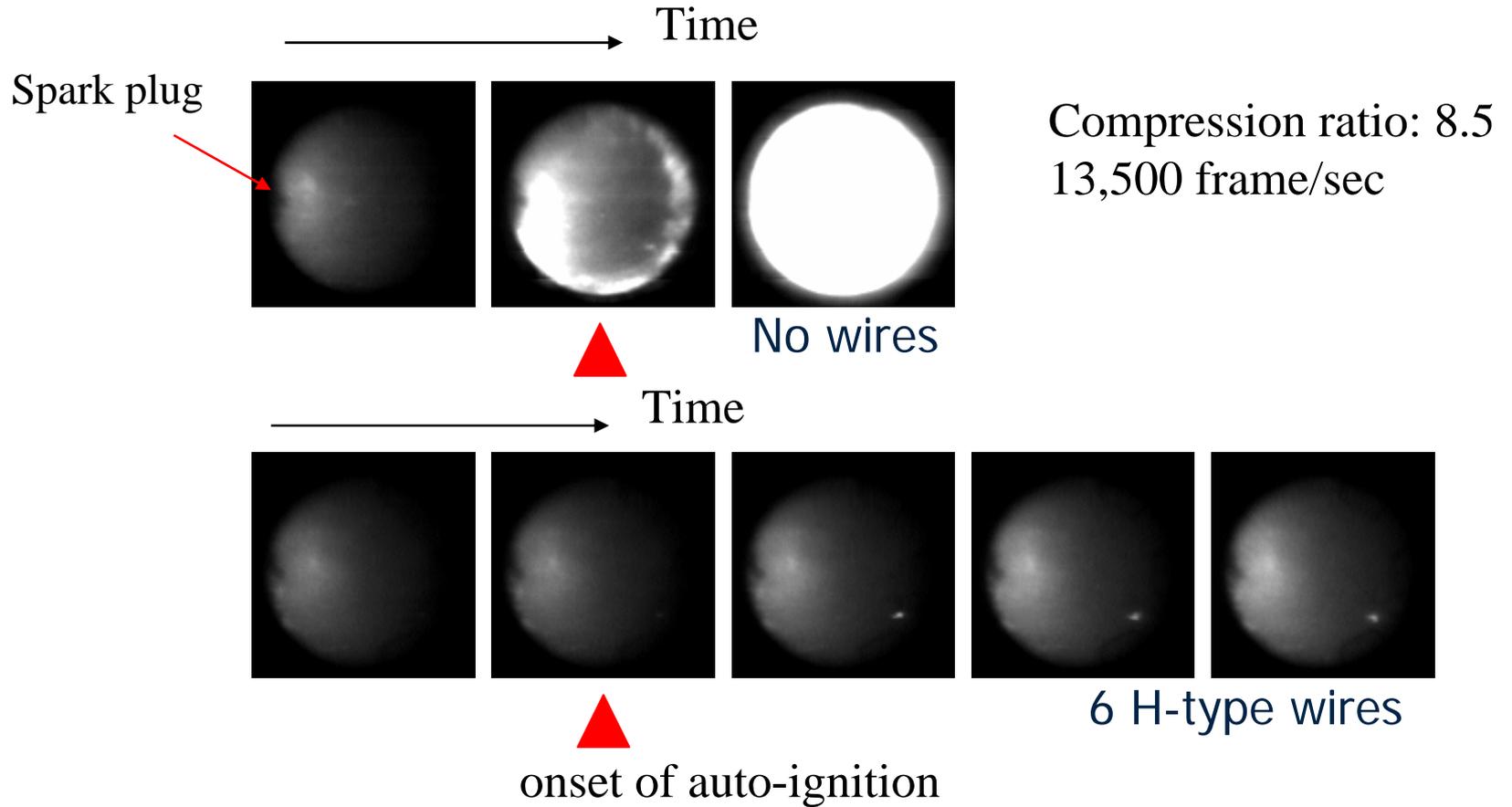
P_e : pressure at combustion end

P_k : pressure at onset of auto-ignition

P_{ig} : pressure at spark ignition

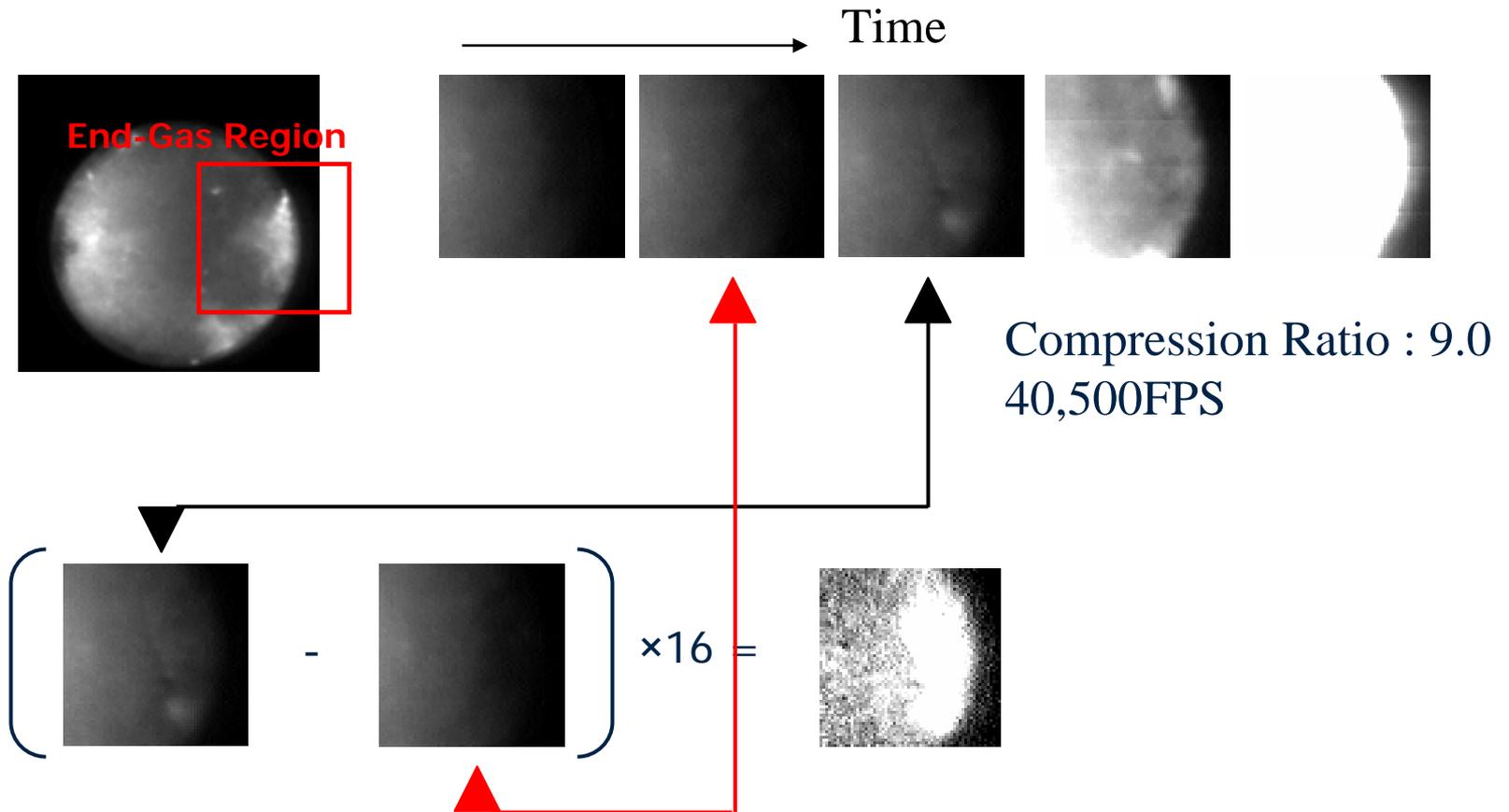


Direct Images just before and after Onset of Auto-ignition



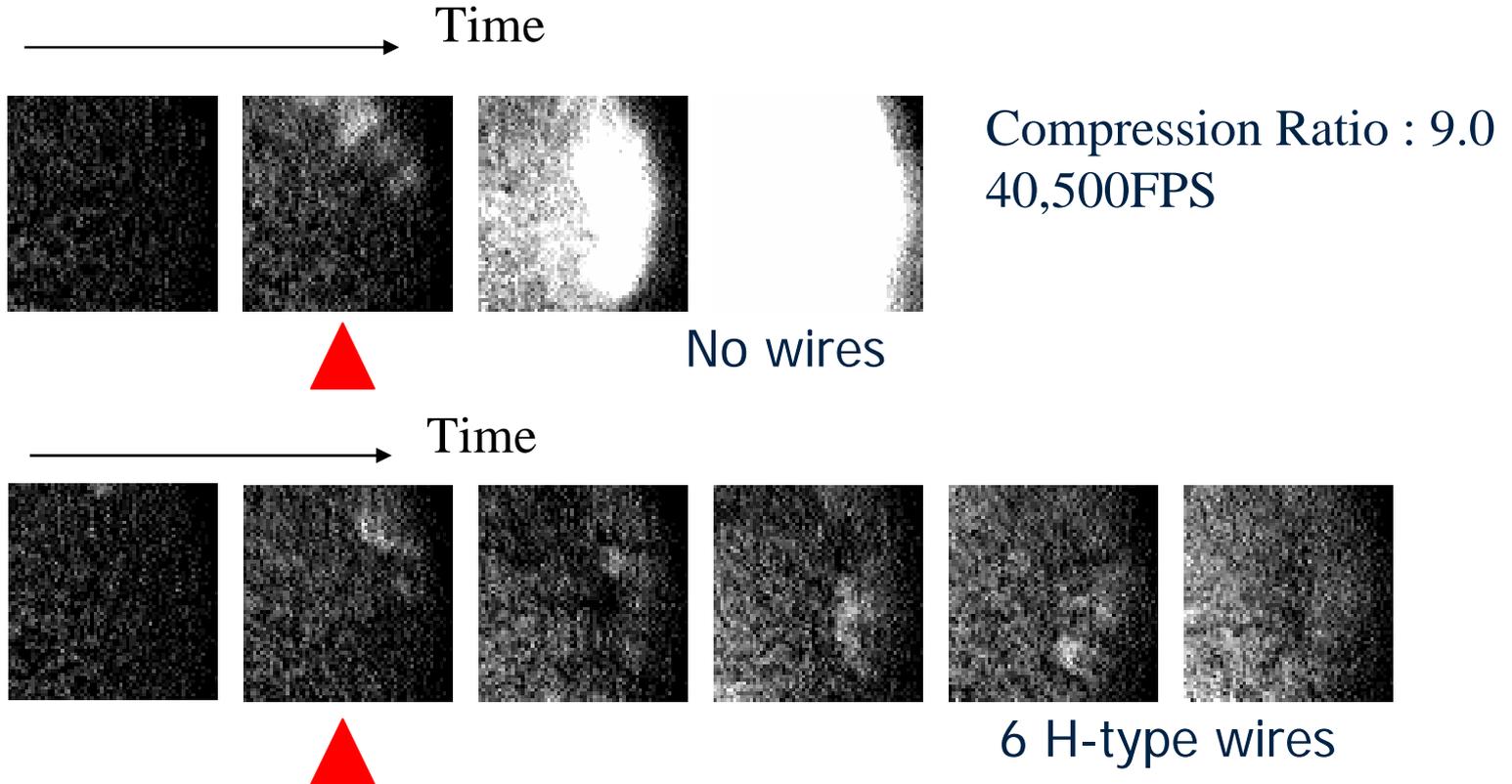


Estimation of Increment of Light Emission Intensity During Time Interval between Two Successive Frames





Time histories of Increment of Light Emission Intensity in End-gas Region



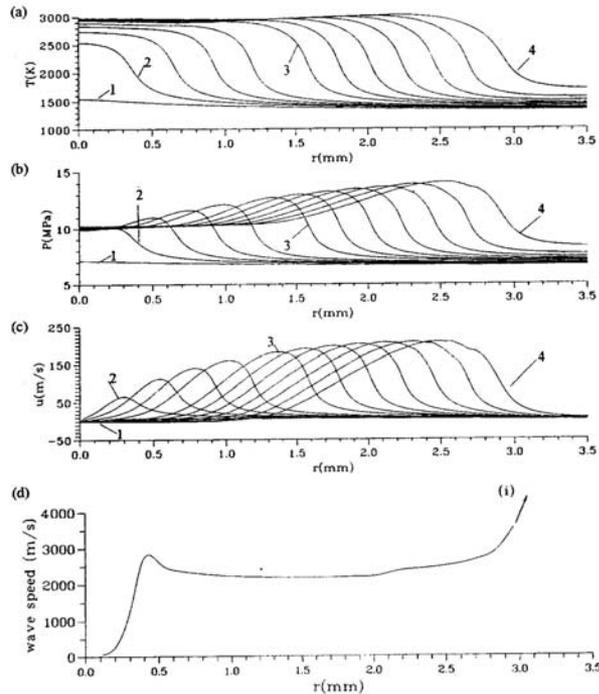
The combustion after auto-ignition becomes 'moderate' with wire installation.



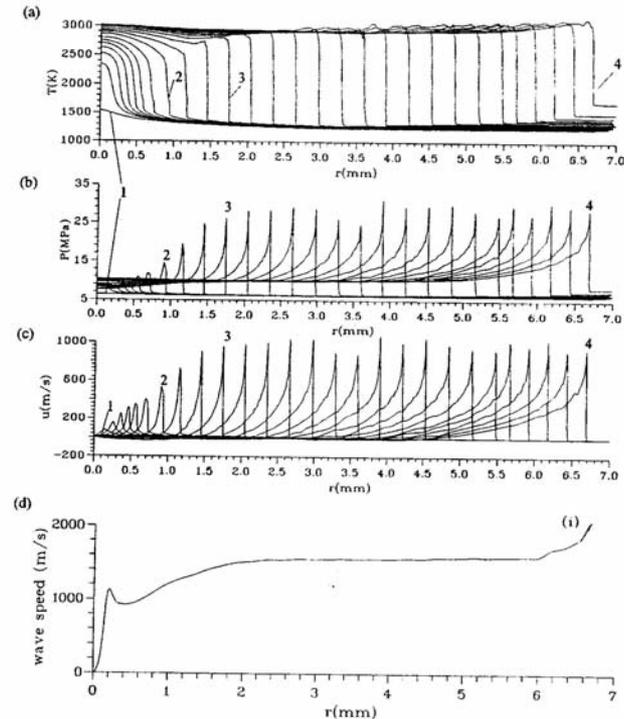
Modes of reaction front propagation (Bradley et al.)



1. Thermal explosion.
2. Supersonic autoignitive deflagration.
3. Developing, developed detonation.
4. Subsonic autoignitive deflagration.
5. Laminar burning deflagration.



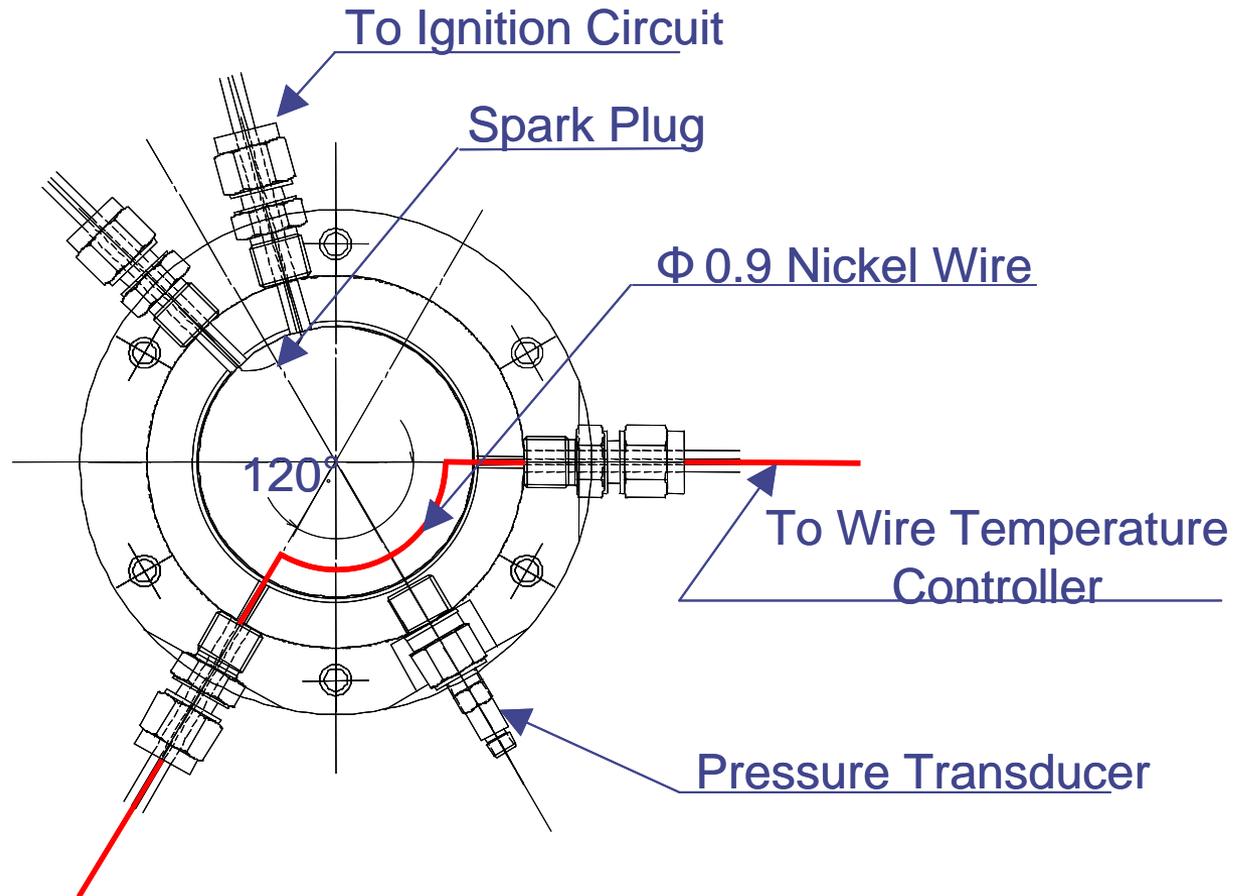
Supersonic autoignitive deflagration



Developing, developed detonation



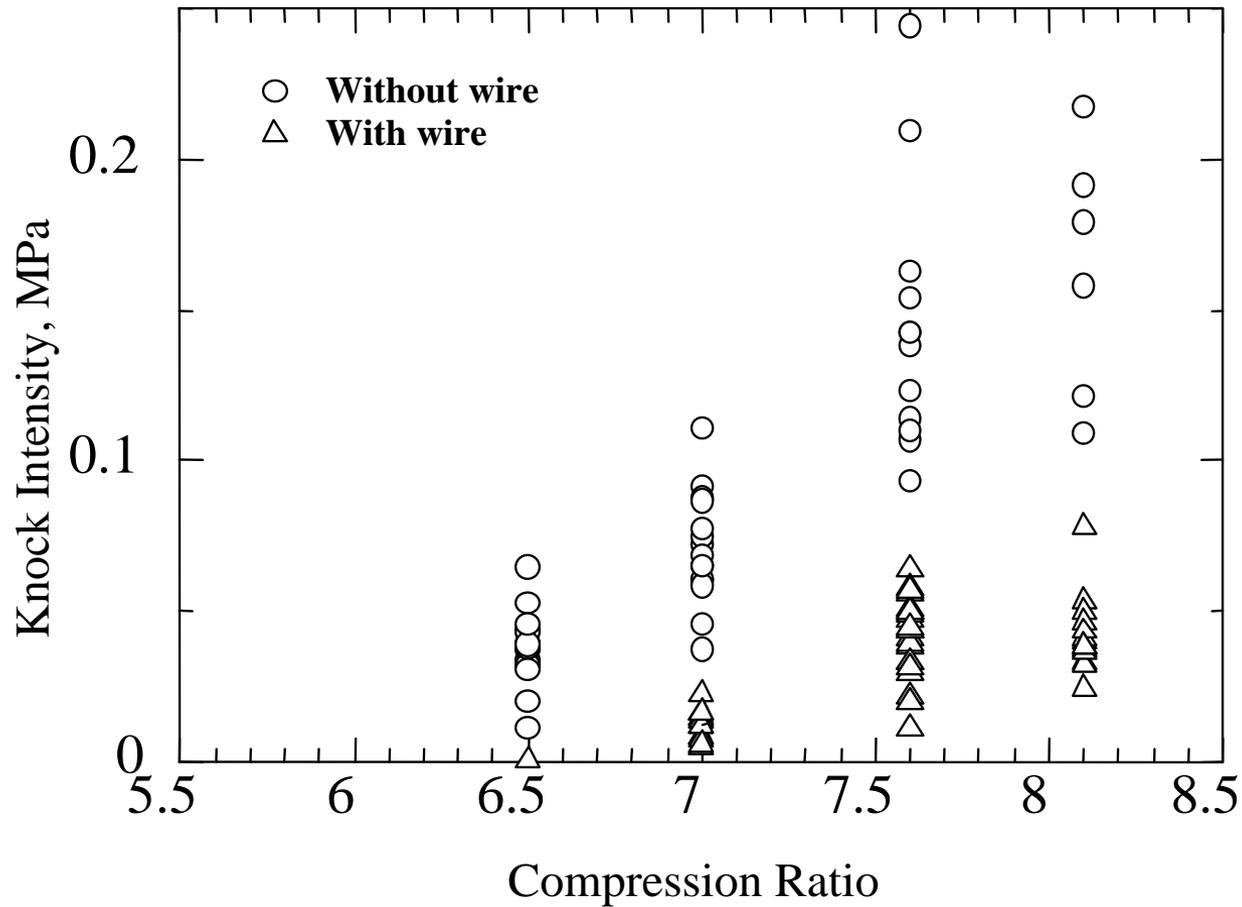
Combustion Chamber (Type-B)



Wire is strung along $1/3$ of the circumference of the cylinder.
Wire temperature can be varied from 298K to 623K



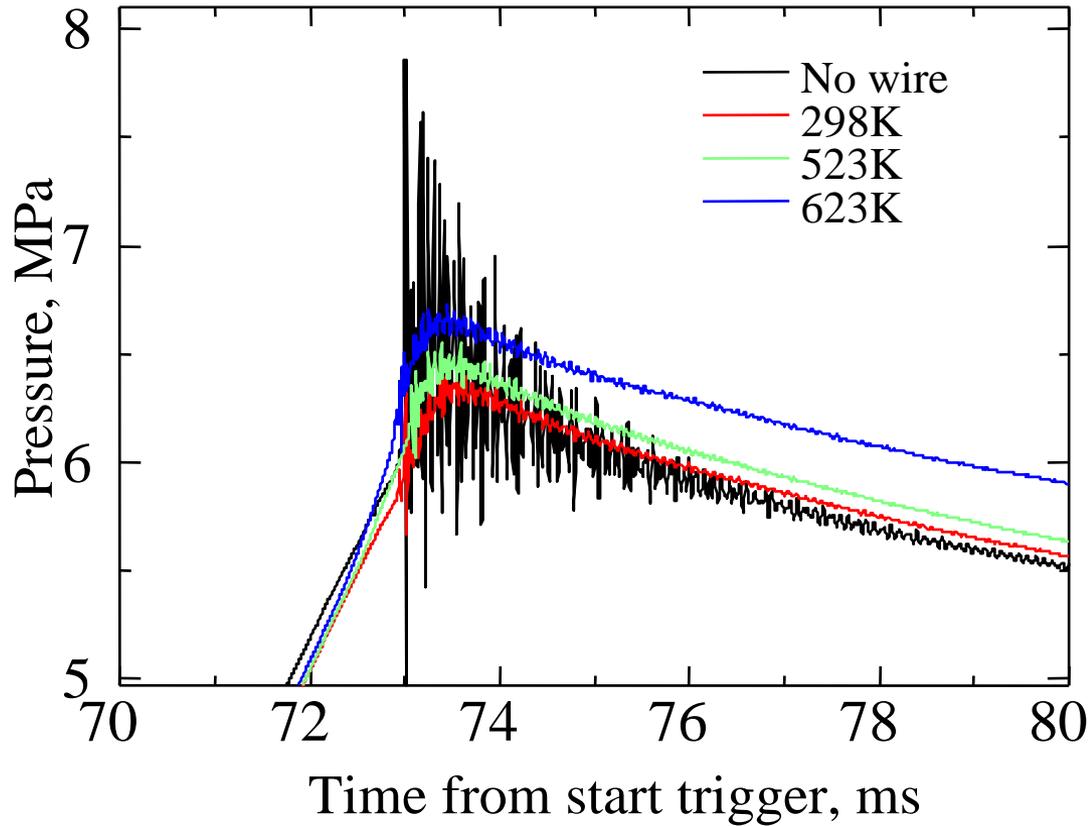
Effect of Wire Installation on Knock Intensity (Type-B, 298 K)





Effect of Wire Temperature on Knock Intensity

Compression ratio: 7.6



Averaged Knock Intensity

No wires:

$KI = 0.1280\text{MPa}$

298K:

$KI = 0.0157\text{MPa}$

523K:

$KI = 0.0109\text{MPa}$

623K:

$KI = 0.0133\text{MPa}$



Conclusions

The significant reduction of the knock intensity is achieved by the wire installation. The knock intensity decreases as the number of the wire increases.

The auto-ignition occurs with and without wire installation.

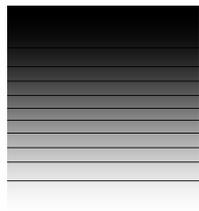
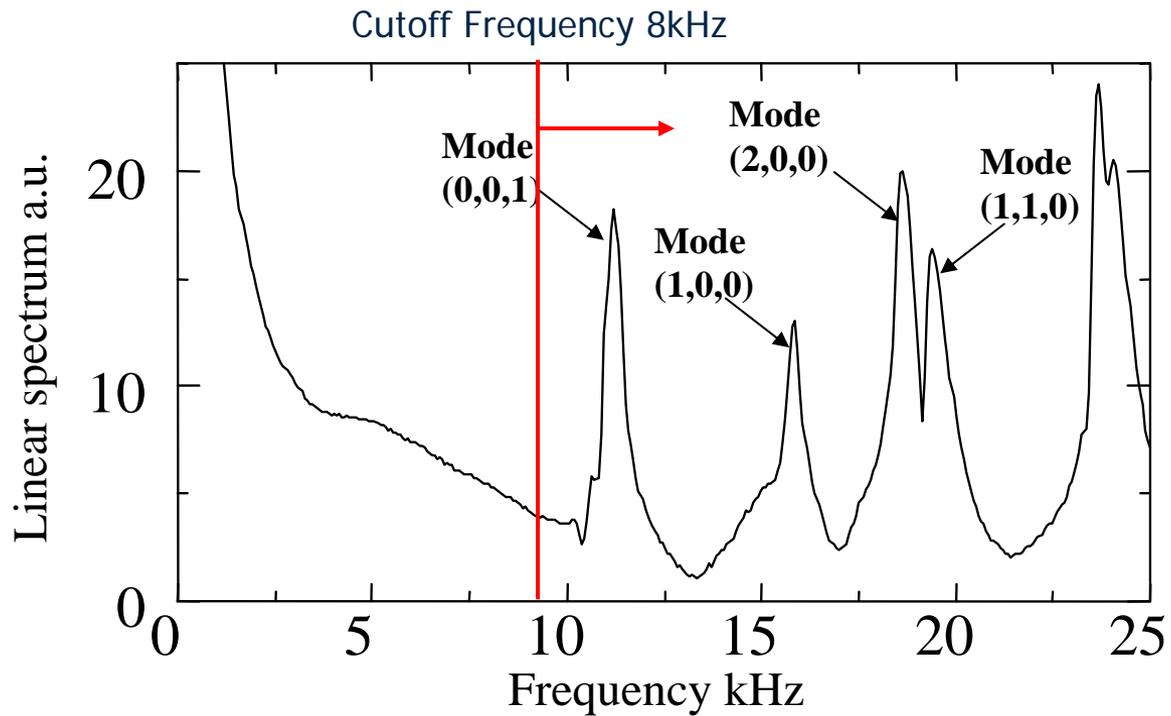
The existence of the wire in the end-gas region has little effect on the normal flame propagation induced by the spark ignition. There is little change in the end gas mass fraction by the wire installation.

The knocking combustion behavior after the auto-ignition becomes moderate with wire installation, which may result in the reduction of the knock intensity.

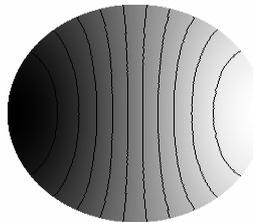
The installation of the wire heated electrically up to 623 K leads to the reduction of the knock intensity.



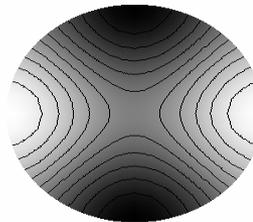
Spectral Power for Pressure Vibration



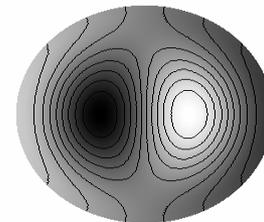
Mode(0,0,1)



Mode(1,0,0)



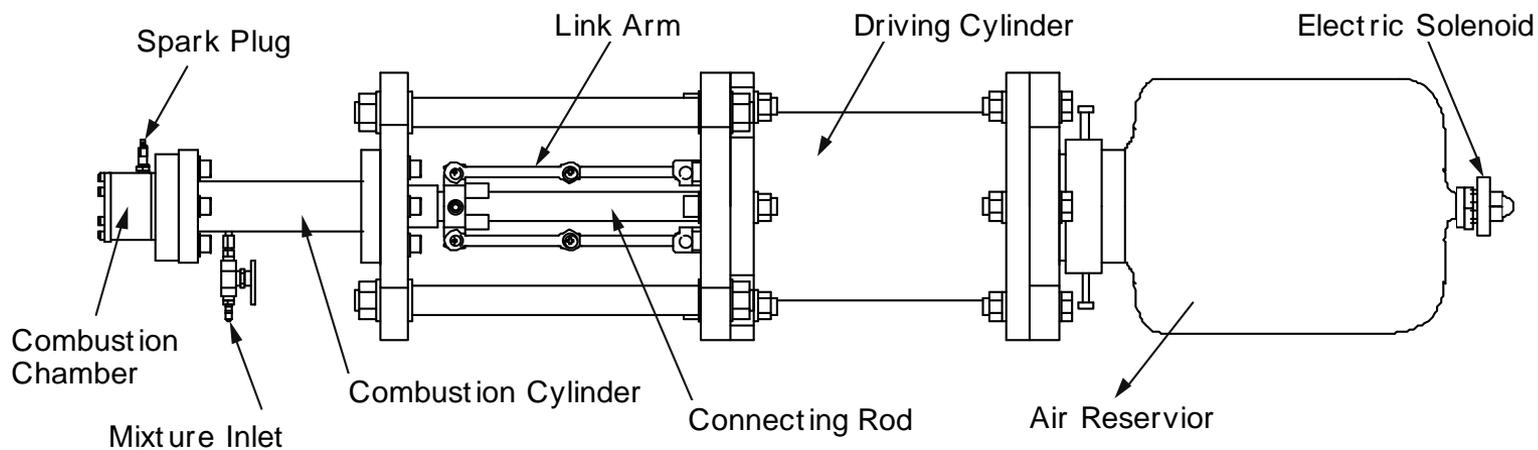
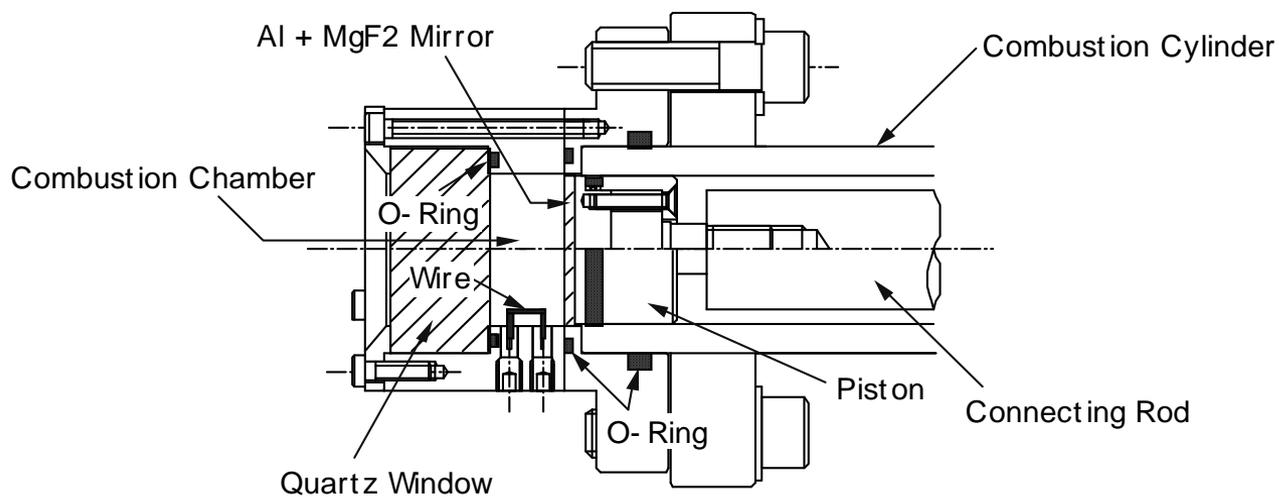
Mode(2,0,0)



Mode(1,1,0)



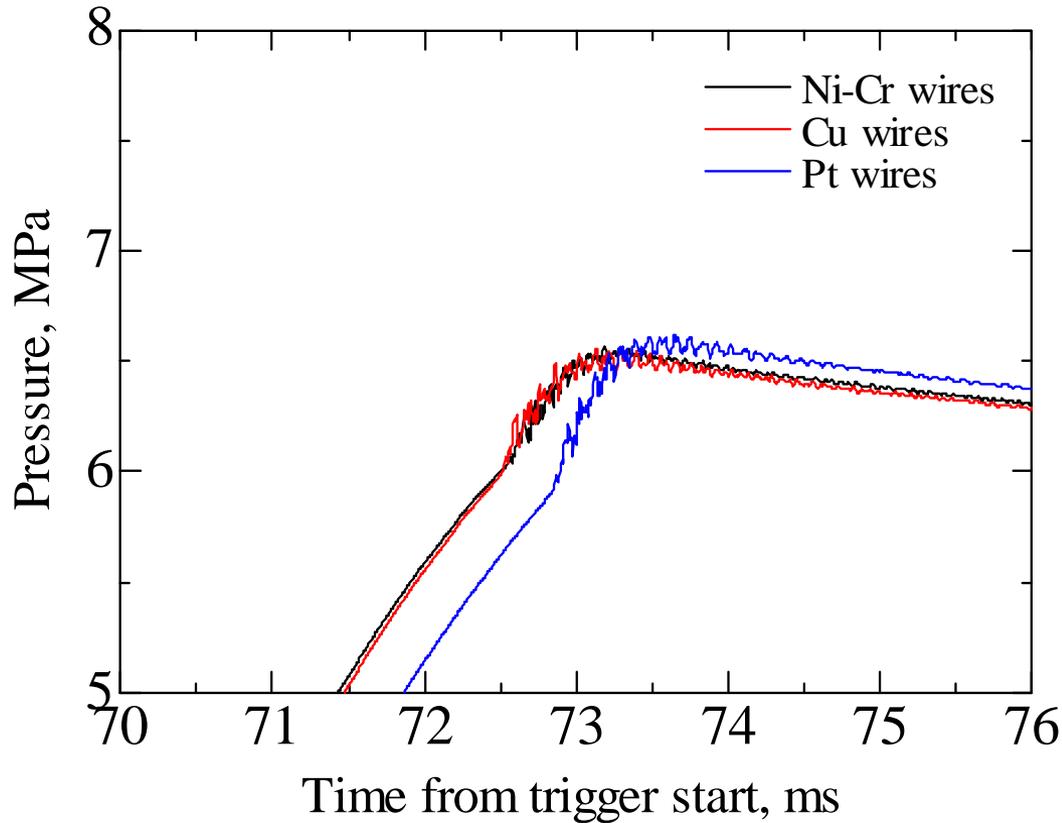
Detailed Configuration of R.C.M.





Effect of Kind of Metal on Knock Intensity

Compression Ratio : 7.5



Averaged knock intensity

- ▶ Ni-Cr wires
KI = 0.064MPa
- ▶ Cu wires
KI = 0.082MPa
- ▶ Pt wires ($\phi=0.7$)
KI = 0.078MPa



SUBTASK 3.1E

EXPERIMENTAL AND NUMERICAL STUDIES OF HIGH INTENSITY COMBUSTION

Contributor: Japan National Committee for IEA/CRD Implementing Agreement on Energy Conservation in Combustion (JECC), Tokyo, Japan.

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Description

The objective of this task is to obtain fundamental information, which can be useful for designing a high intensity compact industrial combustion system. For this purpose, the turbulence-combustion interaction in extremely strong turbulent flow and combustion behavior in the combustion chamber of internal combustion engines have been studied experimentally and numerically.

Two topics on high intensity combustion have been conducted in this subtask. One is the knocking combustion behavior in spark ignition engines, and the other is modeling of turbulent premixed combustion by DNS database.



Simulation of Turbulent Premixed Flames by a Hybrid Turbulence Model

A new turbulence model called hybrid model was proposed to simulate turbulent premixed combustion with flame-generated turbulence and counter-gradient diffusion. The model constants were evaluated by DNS database, and these models were in good agreement with DNS.

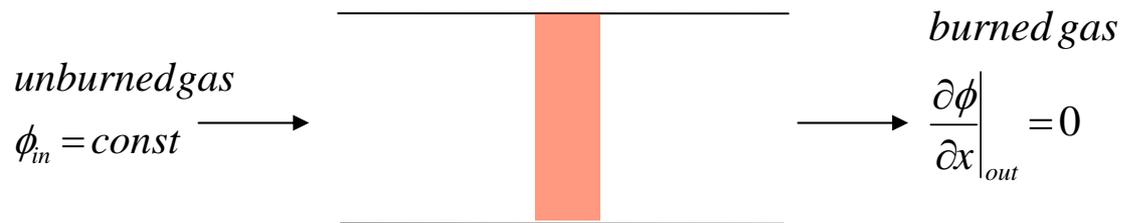


Fig. 1 Simulation model

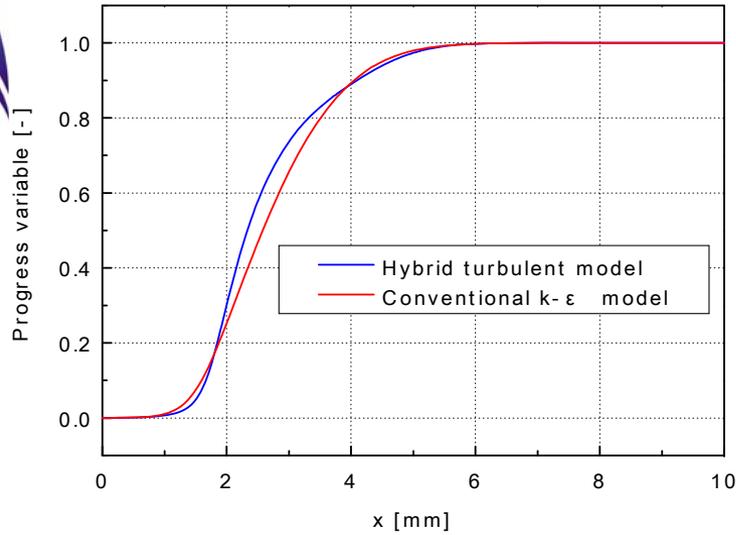


Fig. 2 Distribution of progress variable

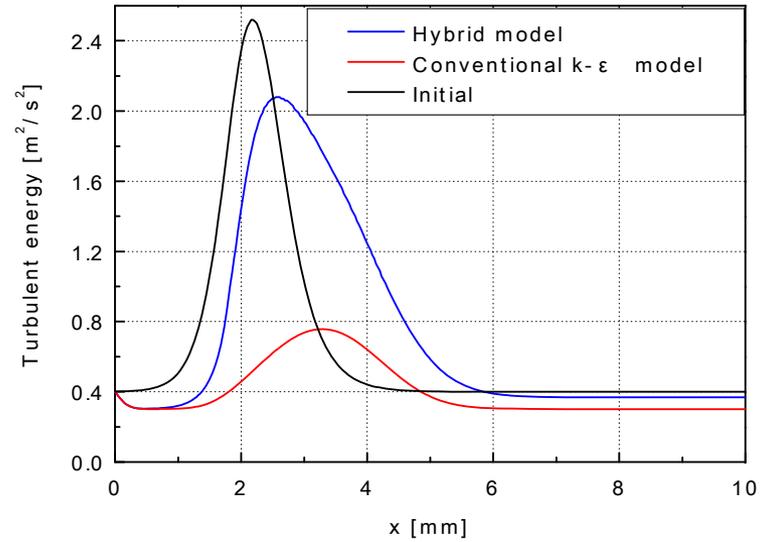


Fig. 3 Distribution of turbulent energy

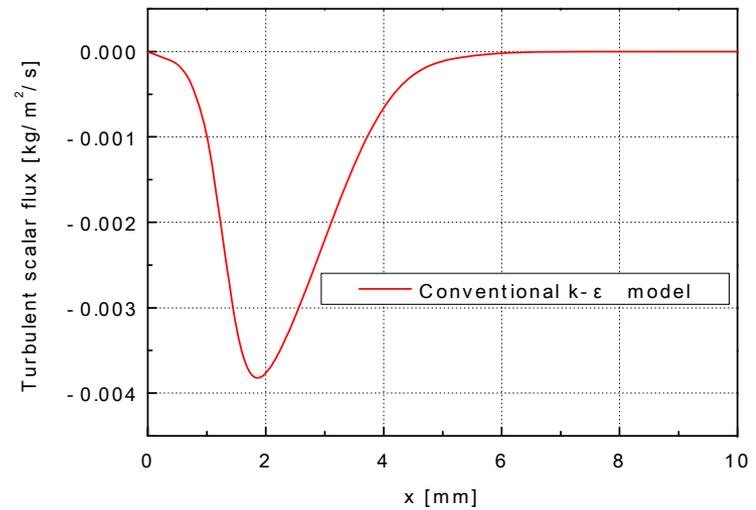
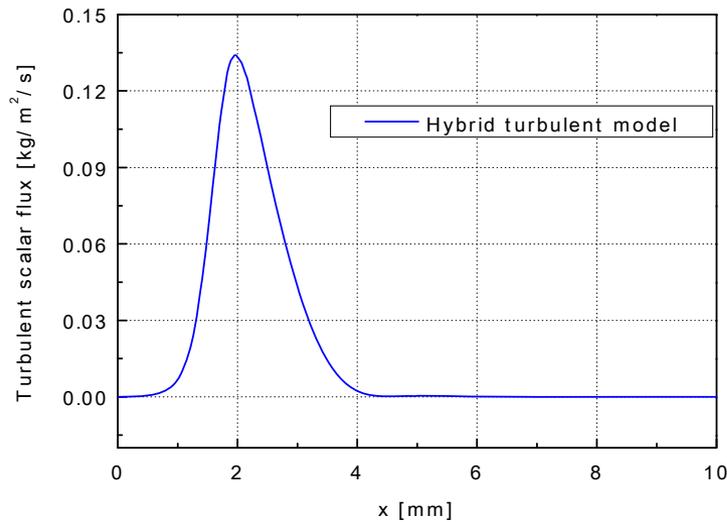


Fig. 4 Distribution of turbulent scalar flux